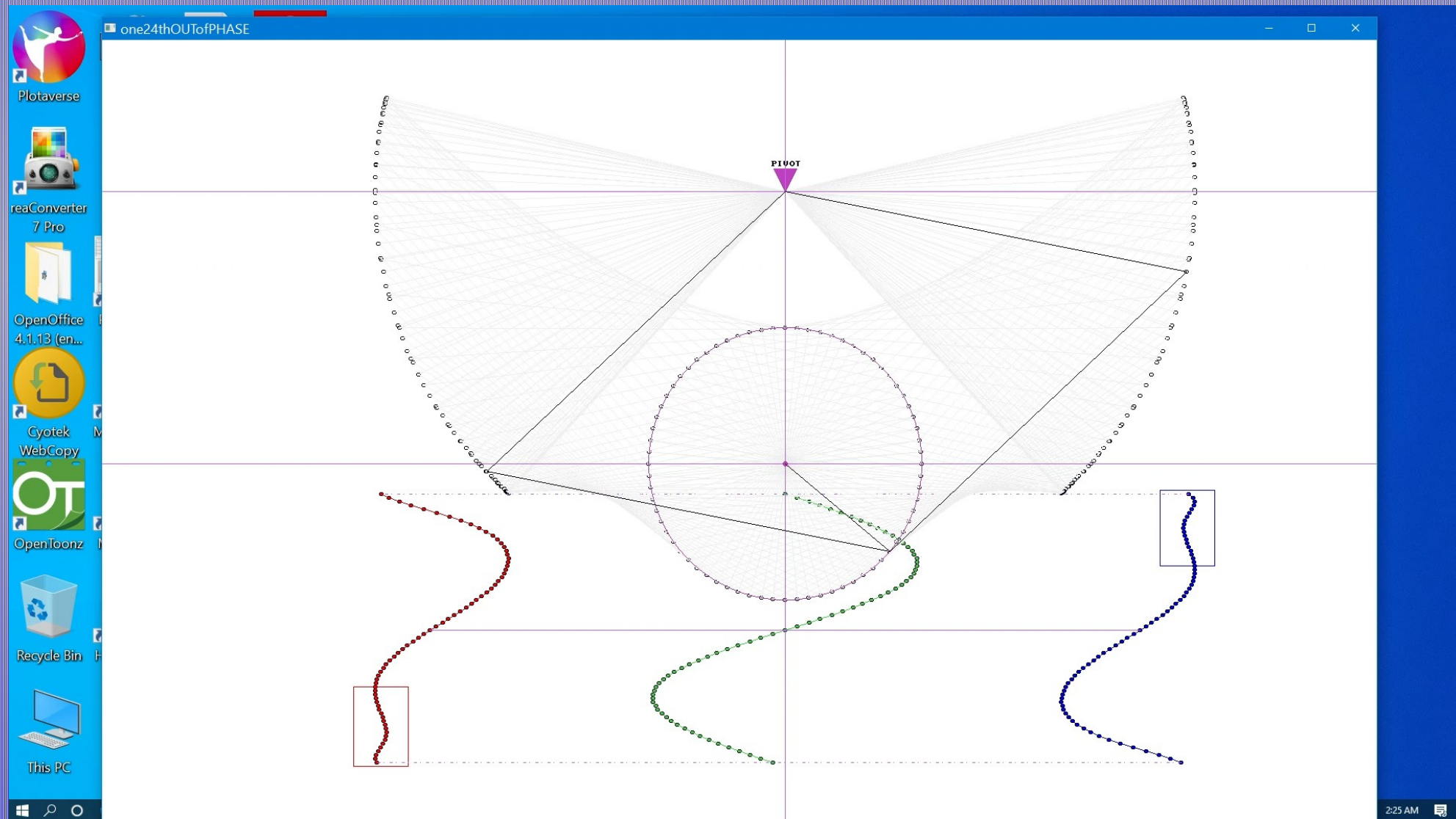


The eb_1/24-out-of-phase_discovery

This be the discovery below :



eb=1/24th-introduction

01=eb BasicIntroductionTutorial=

**What the eb-1/24 out of Phase is, & also
SOME of the Geometry Calculations of it.**

A hardwired physical, **1/24th Out of Phase** relationship, "recently rediscovered", real mechanical action that occur between the **Ordinary Circle** and the **Common Square** when the Circle acting as a rotating Crank, is connected to a Square, whose levers is hinged at the 4 corners so as to be a collapsible Square. This Out of Phase = Out of Step 'jump' is spotted when making a **Comparison** Between the **Left hinge motion Track** to the **Right hinge motion Track**. TO FOLLOW

02=eb BasicIntroductionTutorial=

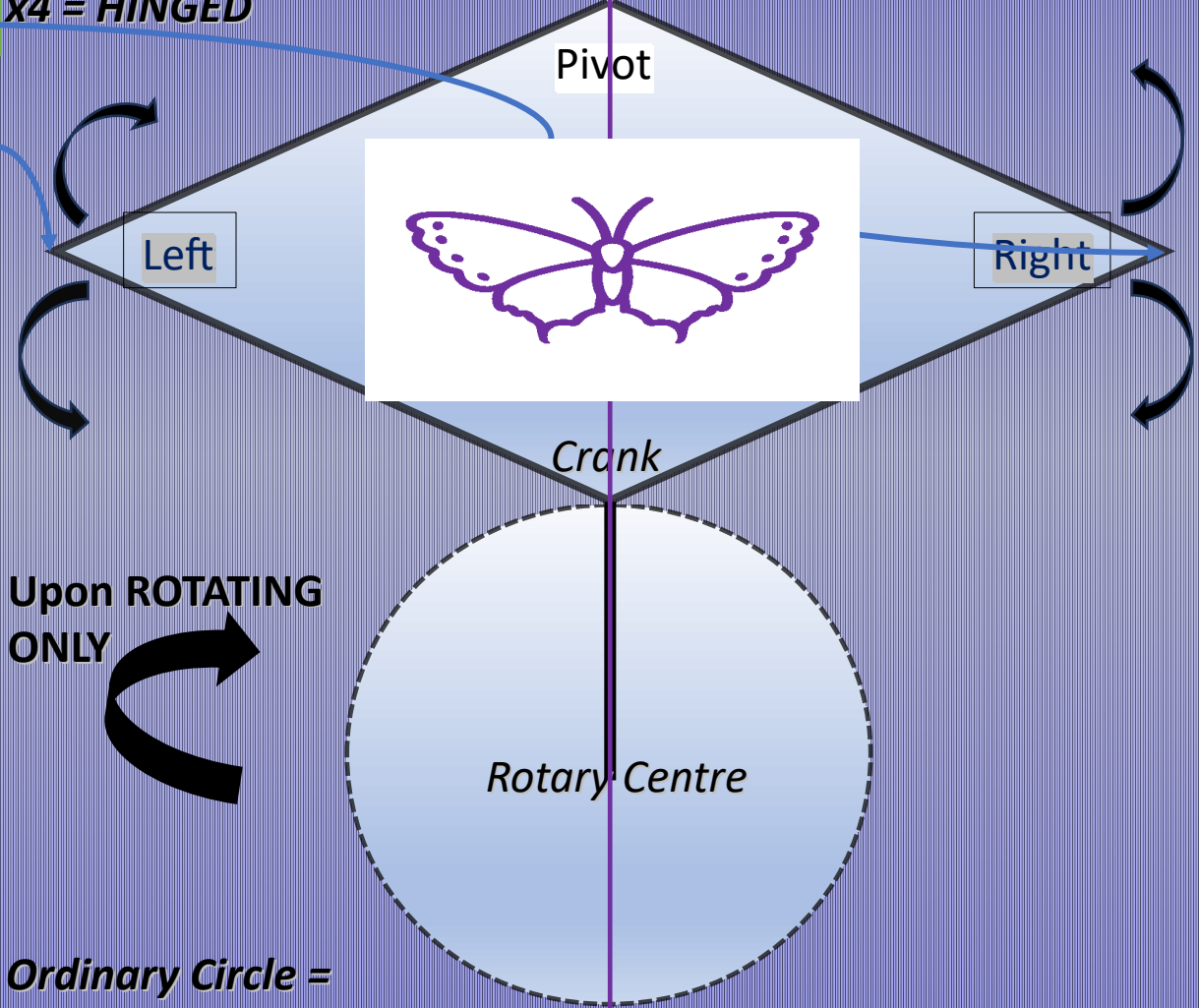
**Physics of Twinesine wave-form / SHAPE
Speculations / Theories, with the eb
Model being the Standard Universal
PARTICLE "Pixel" defining other Objects.**

03=eb BasicIntroductionTutorial=

**Programming modelling to use the eb-1/124,
as well as exploring, the "Fine Butterly
Fold Angle", where Out of Phase
Entanglement Start or Collapse Occur.**

1/24th Out of Phase

Common Square =
x4 = HINGED

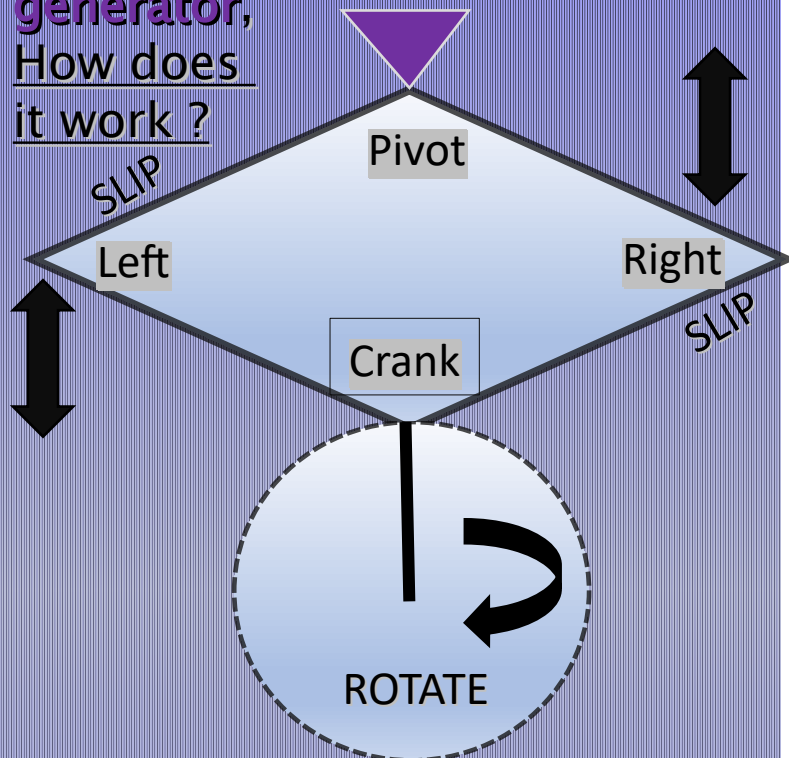


Ordinary Circle =
= CRANK

There EXIST, 3 x Phase Steps = OUT of PHASE & IN PHASE & NOW the
3rd? is the 1/24th OUT of PHASE" or 1/24th to get into IN PHASE
Naturally occurring STABLE Mechanical GEOMETRIC Situation.

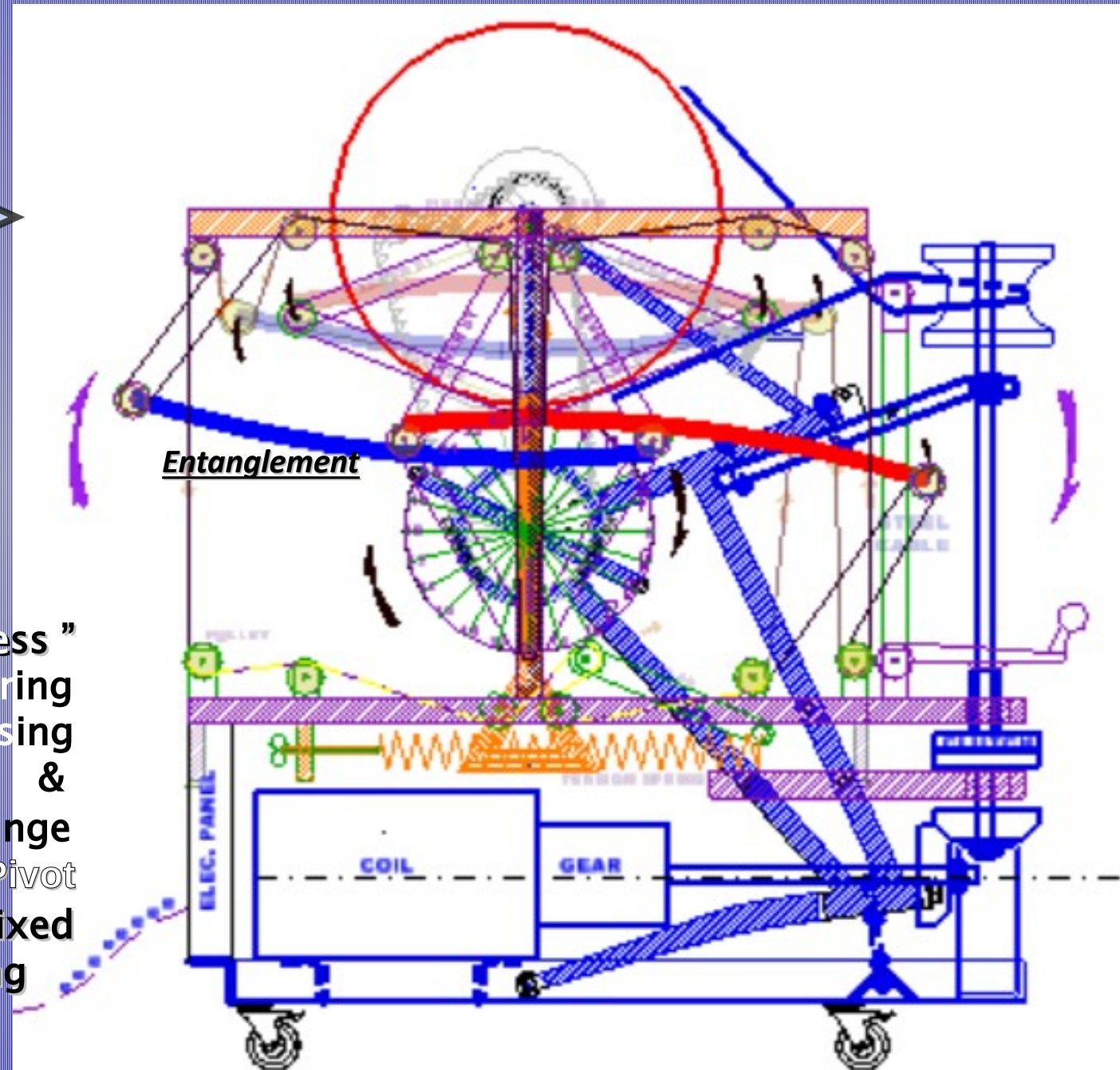
Pg003

The eb
generator,
How does
it work ?



It start with a “Slipperiness” coming into existence, during motion, between the two Opposing Free ends of a Squares Left & Right with the 3rd hinge being a Stationary fixed Pivot & the opposing 4th hinge fixed to a Circle acting as a Moving Cranked Pivot.

1/24th Out of Phase Applied as a Electricity Generator Feeding off from a Trapped Motion , Kinetic Bubble Principle.



1st we need to See prove of the $1/24^{\text{th}}$ phase Difference.

2nd we will see how this small but important Jumping out of phase, is used in a opposite SIDES 2 forces in Unison of rotation Direction, to drive the eb generator.

Right Free point

SLIPPERINESS INDUCED BETWEEN THESE OPPOSING POINTS

PIVOT

Left Free point

UPON

ROTATION

The difference in Phase $1/24^{\text{th}}$ in 180deg.

being just

Rotation of the

Crank from the Top position to the Bottom position & then the same out of step motion

happens on the

upward rotation as well for

the Remaining full

180deg.

The eb generator rotation is thus due to a MECHANICAL EVENT that happens when a collapsible Square is connected to a Crank, acting as the Circle here. See the 2 x free Point Motion Twin Waves traced below. The event is – the Stepping out of Rhythm or out of Phase BETWEEN the Left free point & the Right free point of the Square by $1/24^{\text{th}}$. The Vertical Line through the Stationary Pivot & the Circle Centre is the Line from which the 'Twin' & 1xSine Wave RESULTANT are Plotted.

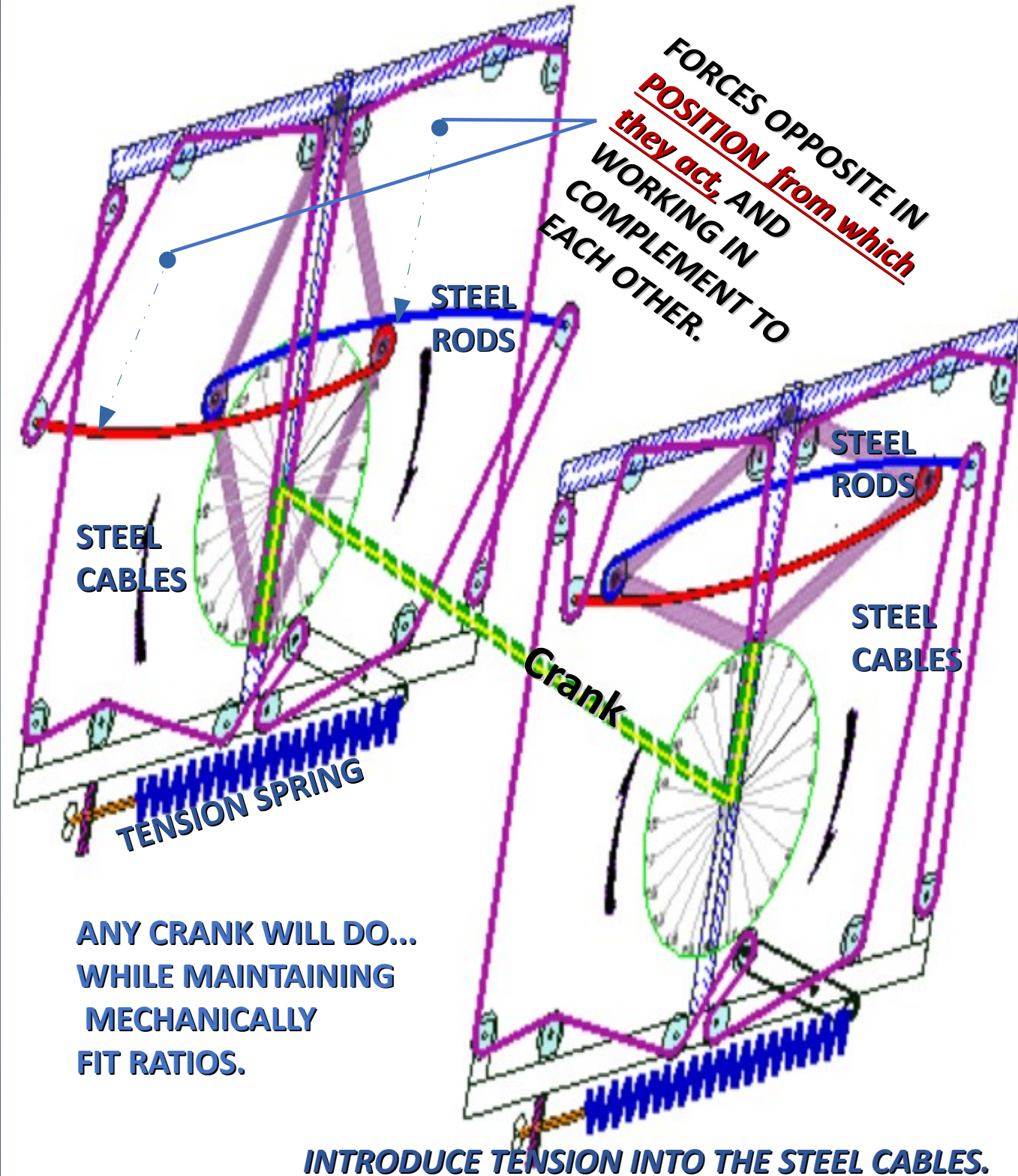
THESE 2 DRAWINGS WITH THE NEXT ONE SHOULD ENABLE YOU TO GO AND BUILD YOUR OWN EB GENERATOR RIGHT NOW. AFTER THE 3RD DRAWING IS ONLY MATHS WITH PHYSICS SPECULATIONS. THE Pythagoras MATHS ARE EASY BUT IT TURNS OUT NASTY AND THE PHYSICS UNFORTUNATELY REQUIRE SOME PRIOR KNOWLEDGE TO MAKE ANY SENSE WHEN YOU ATTACH THESE EB TWINWAVE / TWINESINE STUFF TO IT...

Knowledge of the 1/24th Hardwired Mechanical Slipperiness in the EB setup, then enable us to build the arrangement of Levers, Pulleys and Cables with Springs.

SPRINGS are used for the EB to be Tensioned for the input of POWER /MOTION THAT BECOME ENCASED / TRAPPED, IN THIS eb SET-UP as if it is a Trapped Power IN MOTION BUBBLE.

The Rotary motion that NOW occur Continuously, then is DUE to the UNAVOIDABLE INBUILT MECHANICAL Slipperiness which is used then to turn a generator.

NOTE that the CURVED Steel Rods are transferring Leverage FORCE from Opposite Sides of the EB square and in the Same rotary Direction ALWAYS. THUS, NOT OPPOSING FORCES BUT RATHER FORCES FROM OPPOSING POSITIONS WORKING IN UNISON WITH EACH OTHER.



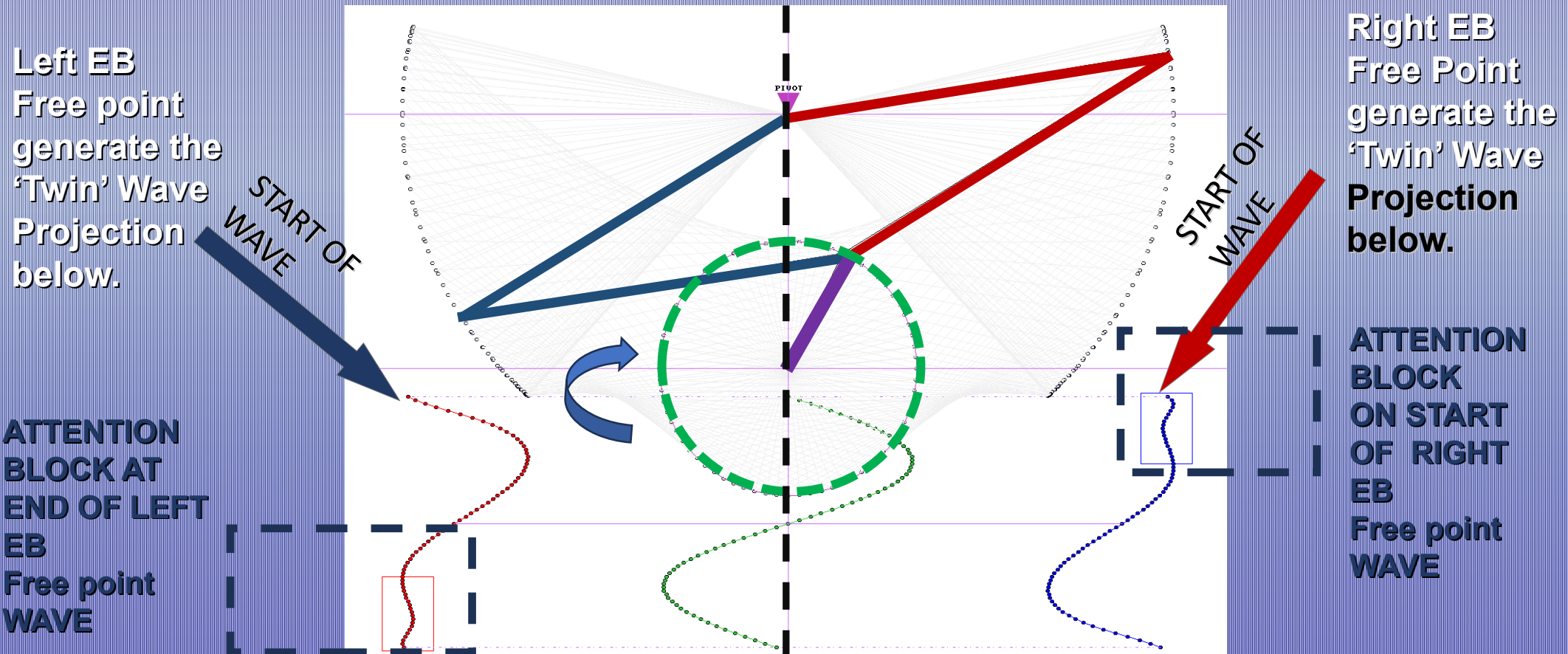
NOW FOLLOWS THE MATHS NEEDED TO UNDERSTAND THE PHYSICS SPECULATIONS LATER.

THE HEAVILY RELIED UPON Pythagoras MATHS ARE INDEED EASY, BUT THE GEOMETRY DOES TURN OUT VERY NASTY. THE PHYSICS ALSO DO REQUIRE SOME BROAD PRIOR KNOWLEDGE TO MAKE ANY SENSE WHEN YOU ATTACH THESE EB TWINWAVE / TWINESINE STUFF TO IT, BECAUSE IT WILL BE TOO MUCH TO EXPLAIN EVERYTHING FROM THE BEGINNING....SO BE FOREWARNED....

1st MATHEMATICAL thing = HOW do you KNOW or PROVE that the claimed $1/24^{\text{th}}$ jump out of phase / slippery Mechanical event REALLY occur , OR HOW TO GEOMETRICALLY CALCULATE IT, SINCE THE EYE LIKELY CANNOT SPOT IT WHEN JUST LOOKING AT THE eb ARRANGEMENT IN MOTION.

1st ANSWER : how to spot the 1/24 out of phase :

THERE ARE at least TWO WAYS TO PROVE THAT THE OUT OF PHASE JUMP EXIST, ONE BEING THE 'HARD' GEOMETRIC WAY OF PLOTTING THE WAVES FROM A HAND DRAWN OR CAD CONSTRUCTION WHICH IS TEDIOUS (the Geometry below uses $360/5 = 72$ points to plot for each wave x3) Phew!!! Naturally a coarser step of $360/15 = 24$ is easier AND WELL WORTH THE EFFORT EVEN IF YOU ONLY DO IT ONCE. NOW NOTE THE ATTENTION BLOCKS AT THE WAVES SINCE IT IS EASIER TO SPOT THE JUMP BY COMPARING THE END & START POINTS OF THE OPPOSING 2x "MIRRORED" WAVES.

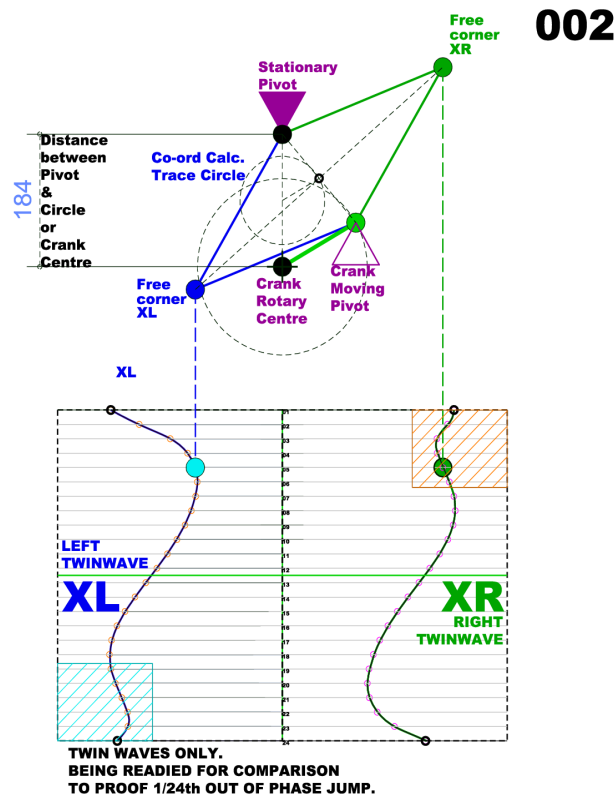


@ 1st GLANCE ONE WAVE LOOK LIKE A FLIPPED MIRROR OF THE OTHER = NOT ? TRUE...!

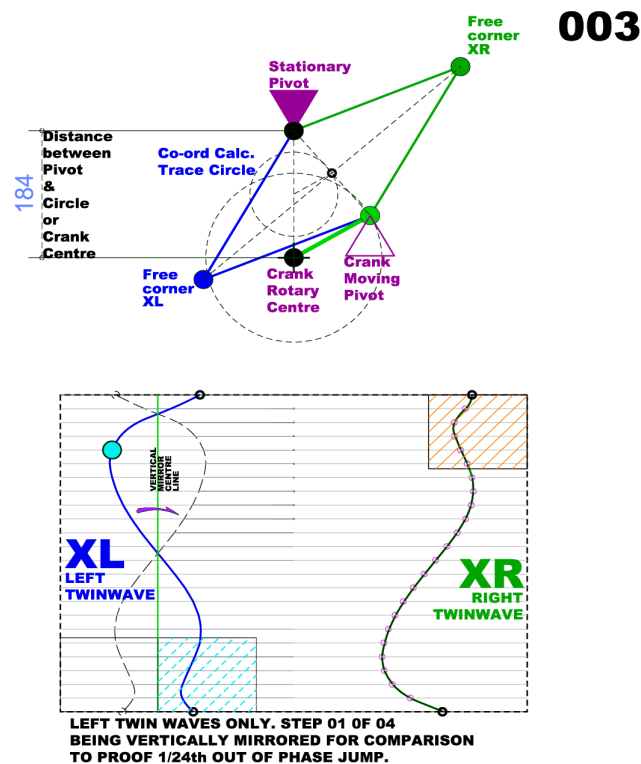
1st ANSWER continued :-how to spot the 1/24 out of phase.

By COMPARING the TWINWAVES GRAPHICALLY and see how they DO NOT FIT IN THEIR NATURAL STATE.

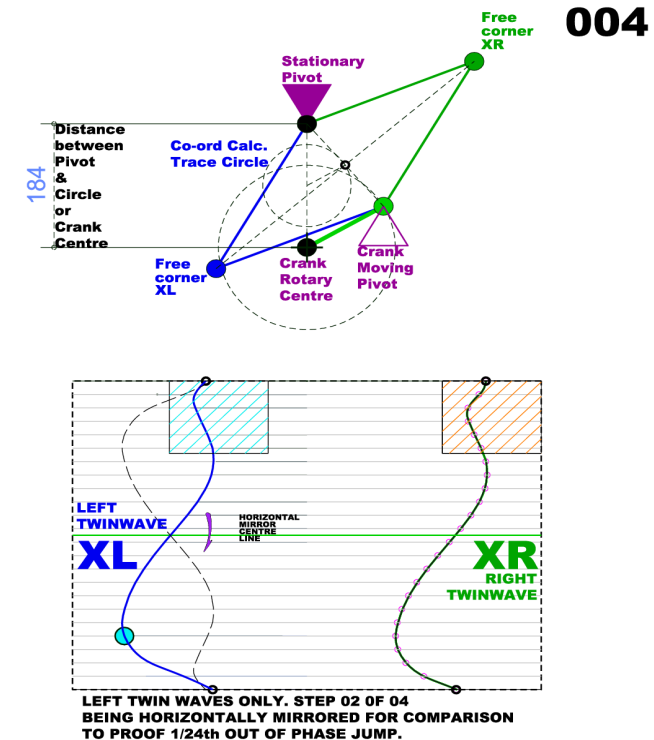
So why call them Twins if they do not exactly look alike = only when you SHIFT one Wave by 1/24th do it EXACTLY FIT ON TOP OF THE OTHER WAVE, With the reality being that they are ONLY TWINS for 23xParts *WHEN* OVERLAID for Comparison purpose.
Lets do some Flipping & Mirroring below....



**1/24th OUT OF STEP PROOF:-
THE DIFFERENCE IN WAVES
IN THEIR NATURAL STATE**

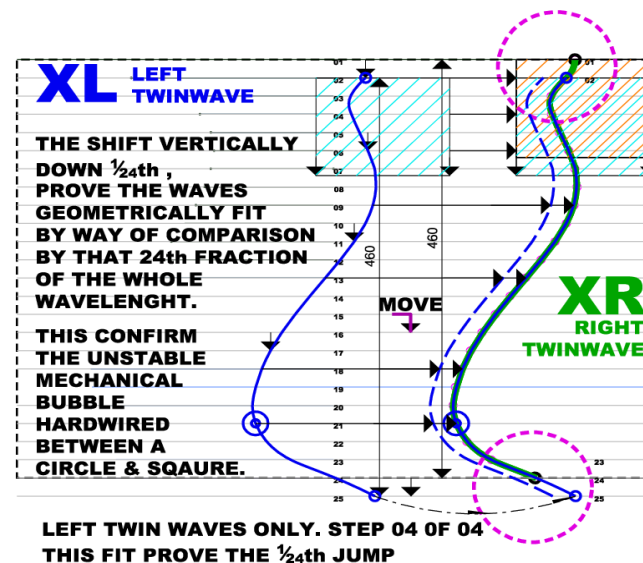
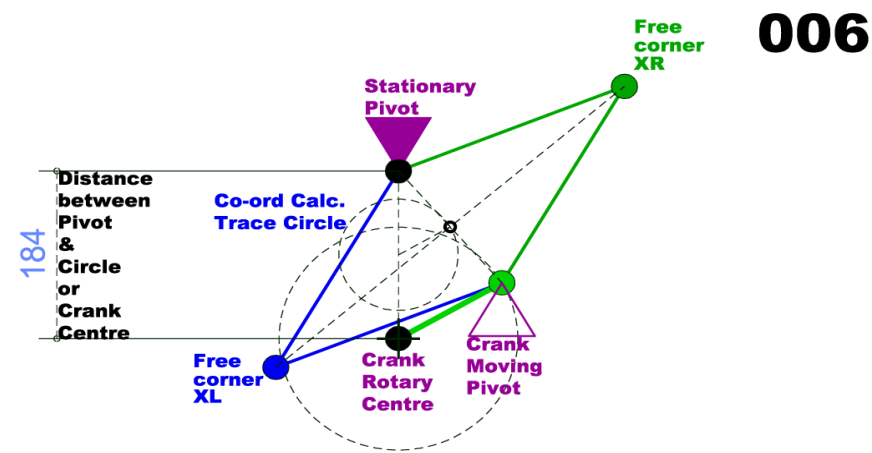
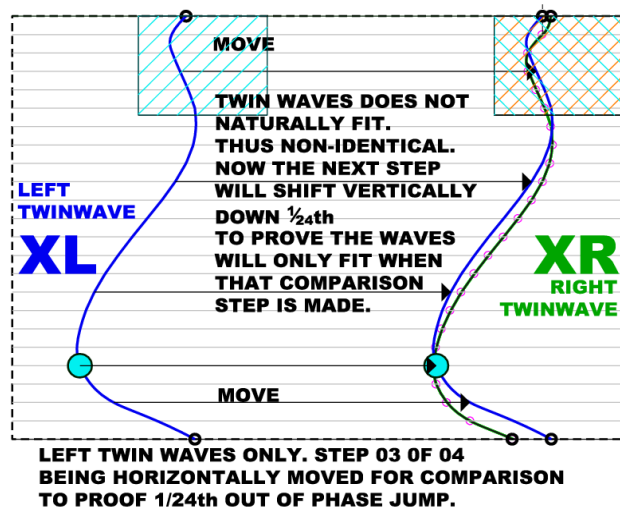
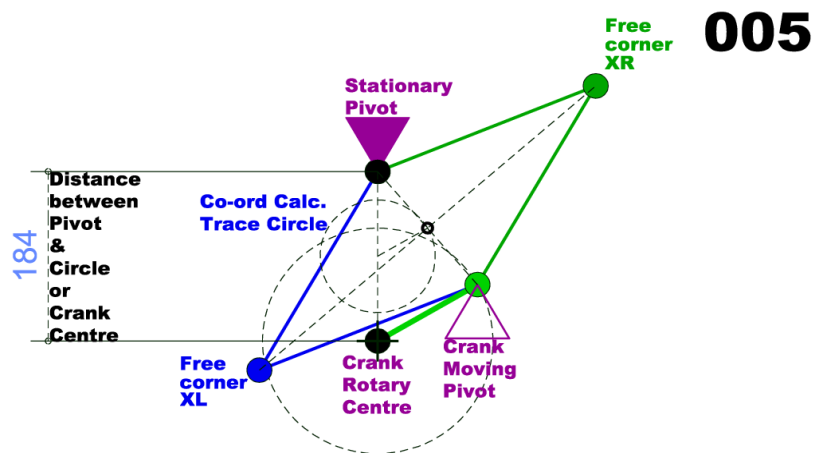


**1/24th OUT OF STEP PROOF:-
SIMPLIFYING BY VERTICALLY
FLIPPING ONE WAVE**



**1/24th OUT OF STEP PROOF:-
HORIZONTALLY FLIPPING
THE FLIPPED WAVE**

Let's move the Flipped & Mirrored LEFT wave Over the RIGHT twin wave to see in panel005, that they do not exactly match. NOW NOTE: In panel006 we move the Flipped & Mirrored LEFT wave DOWN by $1/24^{\text{th}}$ of the GRID and overlay again. NOW THE 23 PARTS OF THE WAVES FIT PERFECTLY OVER EACH OTHER PROVING THE OUT OF PHASE-NESS IN THEIR NATURAL STATE



**1/24th OUT OF STEP PROOF:-
COMPARING THE FLIPPED
WAVE TO REVEAL MISMATCH**

**1/24th OUT OF STEP PROOF:-
FLIPPED MOVE DOWN BY
1/24th TO MATCH XR**

Let's apply the HARD-WIRED SLIPPAGE below. REMEMBER THAT WE ARE NOT DOING MATH. YET....JUST GEOMETRIC COMPARISONS.

008

A THE TWIN WAVES IN THEIR NATURAL GENERATION ARE NON-IDENTICAL ACROSS THEIR WHOLE LENGTH WHEN COMPARED TO EACH OTHER WITH ONE ONLY, FLIPPED VERTICALLY AND HORIZONTALLY.

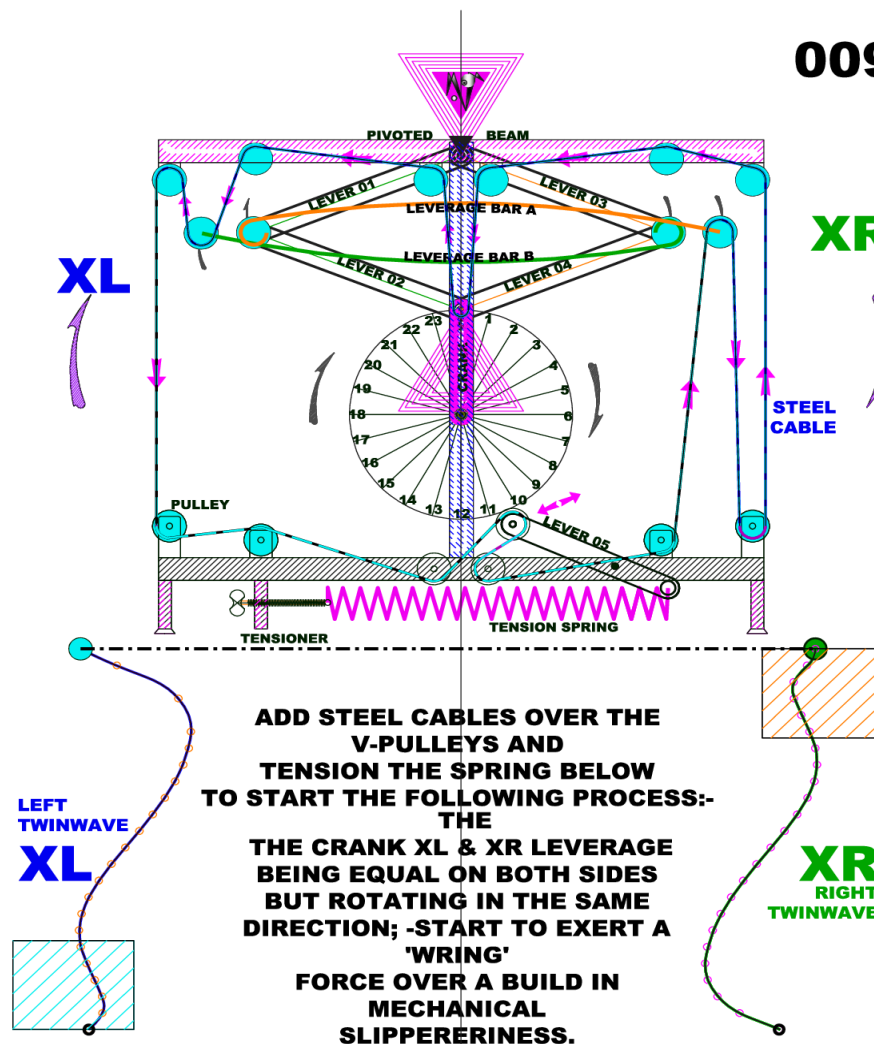
B HOWEVER WHEN ONE FLIPPED NON-IDENTICAL TWIN WAVE SHIFTS BY $\frac{1}{24}$ th OF IT'S WAVELENGTH ALONG IT'S WAVE PATH , THEN AN EXACT FITTING OVER THE COMPARISON TWINWAVE. CAN BE MADE.

C THIS PROOF THAT A 1/24th OUT OF STEP OR OUT OF PHASE MECHANICAL NATURAL DIFFERENCE EXIST IN THIS PARTICULAR SQUARE + CIRCLE SYSTEM, THAT IS BETWEEN THE 2 FREE CORNERS OF THE SQUARE THAT "JUMPS" OUT OF BALANCE FROM EACH OTHER DURING 2 HALF'S AT THE MOMENT ROTARY MOTION IS INTRODUCED.

D THE EB GENERATOR MECHANICAL ARRANGEMENT IS ABOUT PRACTICAL APPLICATION OF THIS $\frac{1}{24}$ th MECHANICAL HARDWIRED TRAPPED INSTABILITY.

1/24th OUT OF STEP WITH TWINWAVES ENLARGED. THE FOCUS IS NOW ON EB DRIVE.

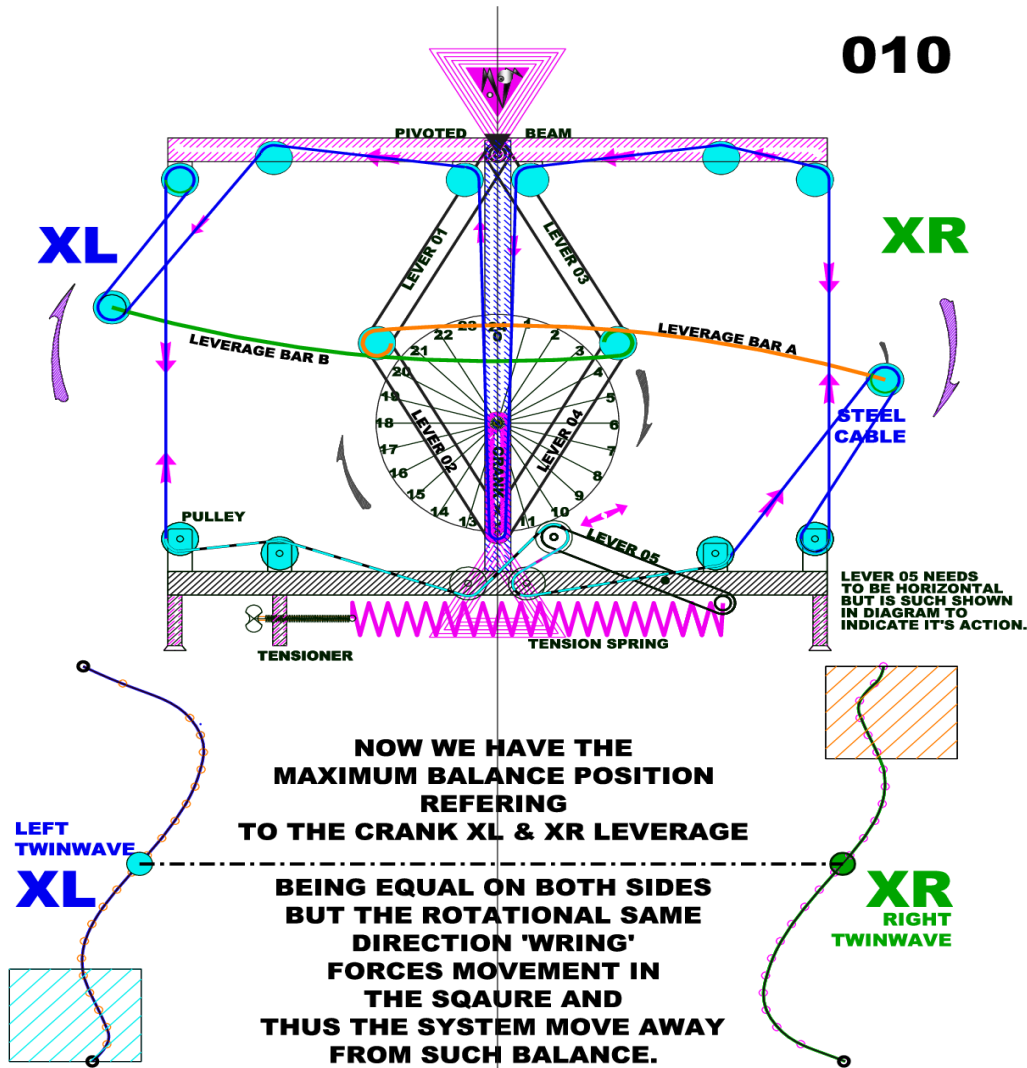
009



1/24th OUT OF STEP USE: TOP OF CRANK POSITION.

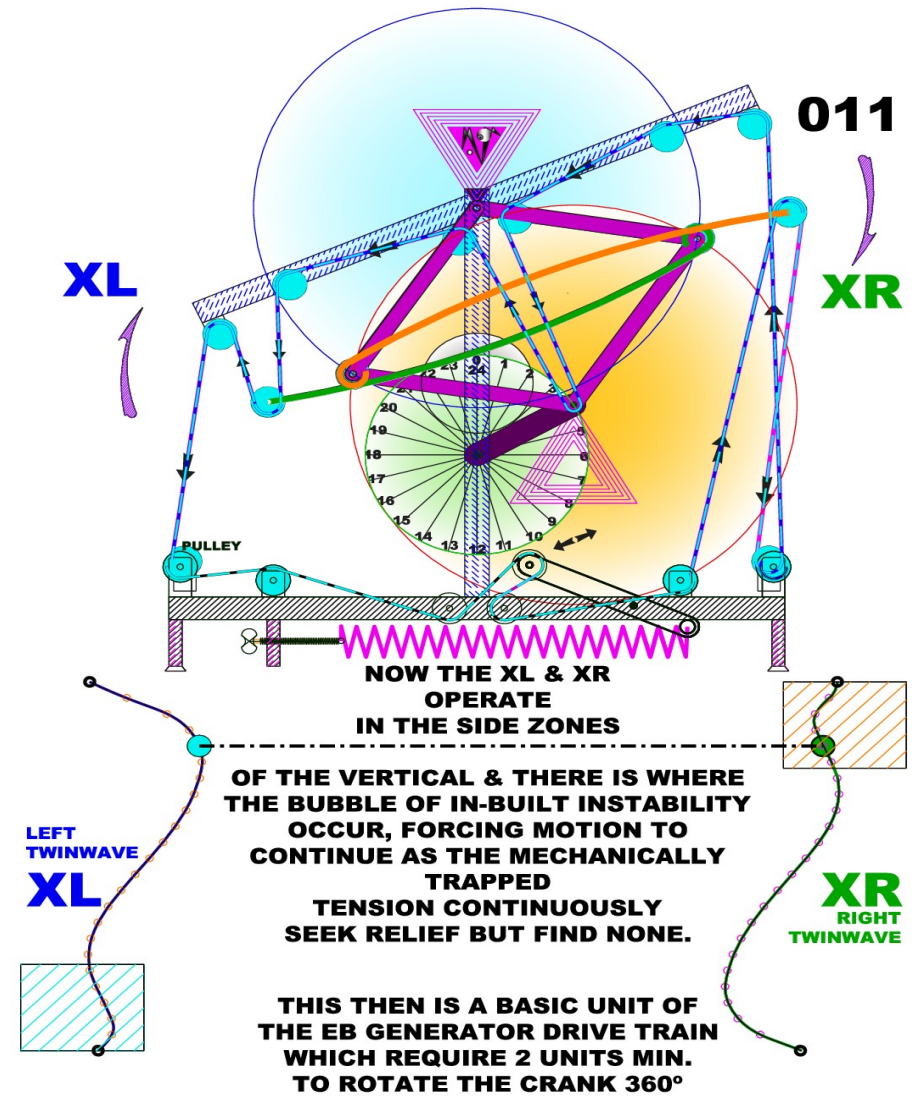
NOW WE APPROACH THE MATHEMATICS OF FB.

010



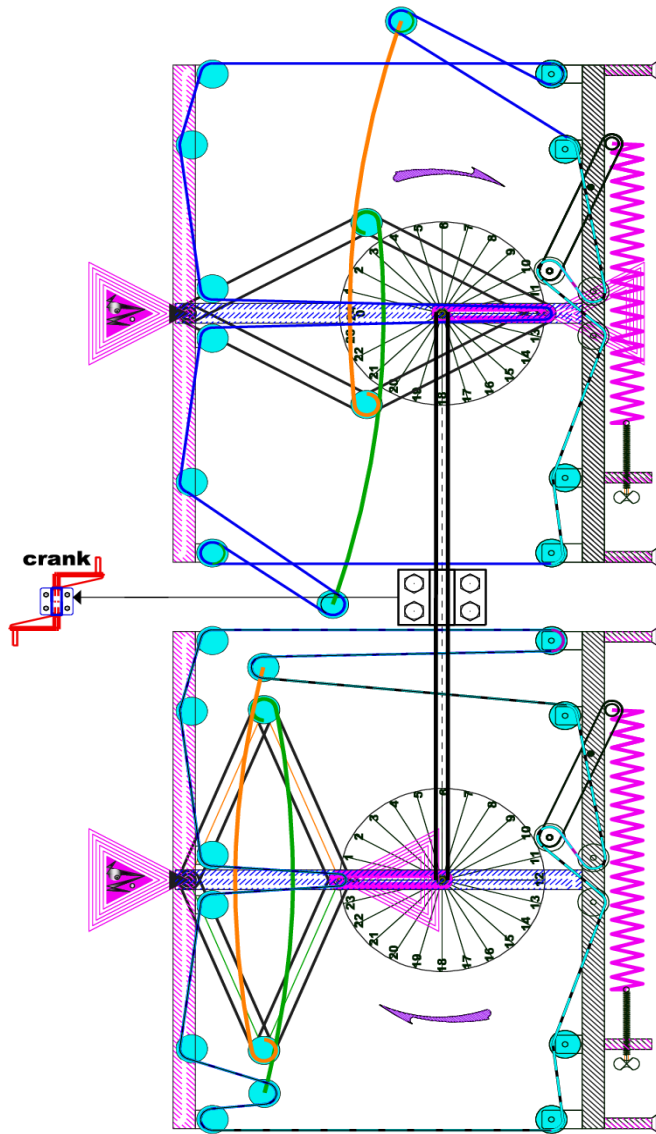
**1/24th OUT OF STEP USE:-
BOTTOM OF CRANK**

011



**1/24th OUT OF STEP USE:-
HALF OF EB DRIVE TRAIN**

VERY NEAR THE MATHEMATICS OF EB NOW.....



1/24th EB FULL DRIVE TRAIN.

THIS THEN IS A
MINIMUM
WORKABLE
UNIT OF THE
EB GENERATOR
DRIVE TRAIN
WHICH WILL
UPON
SIMULTANEOUS
TENSIONING
DRIVE THE
CRANKSHAFT
THRU 360°
CONTINUOUSLY
SINCE THERE
IS NO RELAX
OR DISSIPATION
OF THE 1/24th
MECHANICAL
BUBBLE OF
HARDWIRED
INSTABILITY UNTO
WHICH TORQUE
KEEP BEING
SPRING
TENSION
APPLIED.

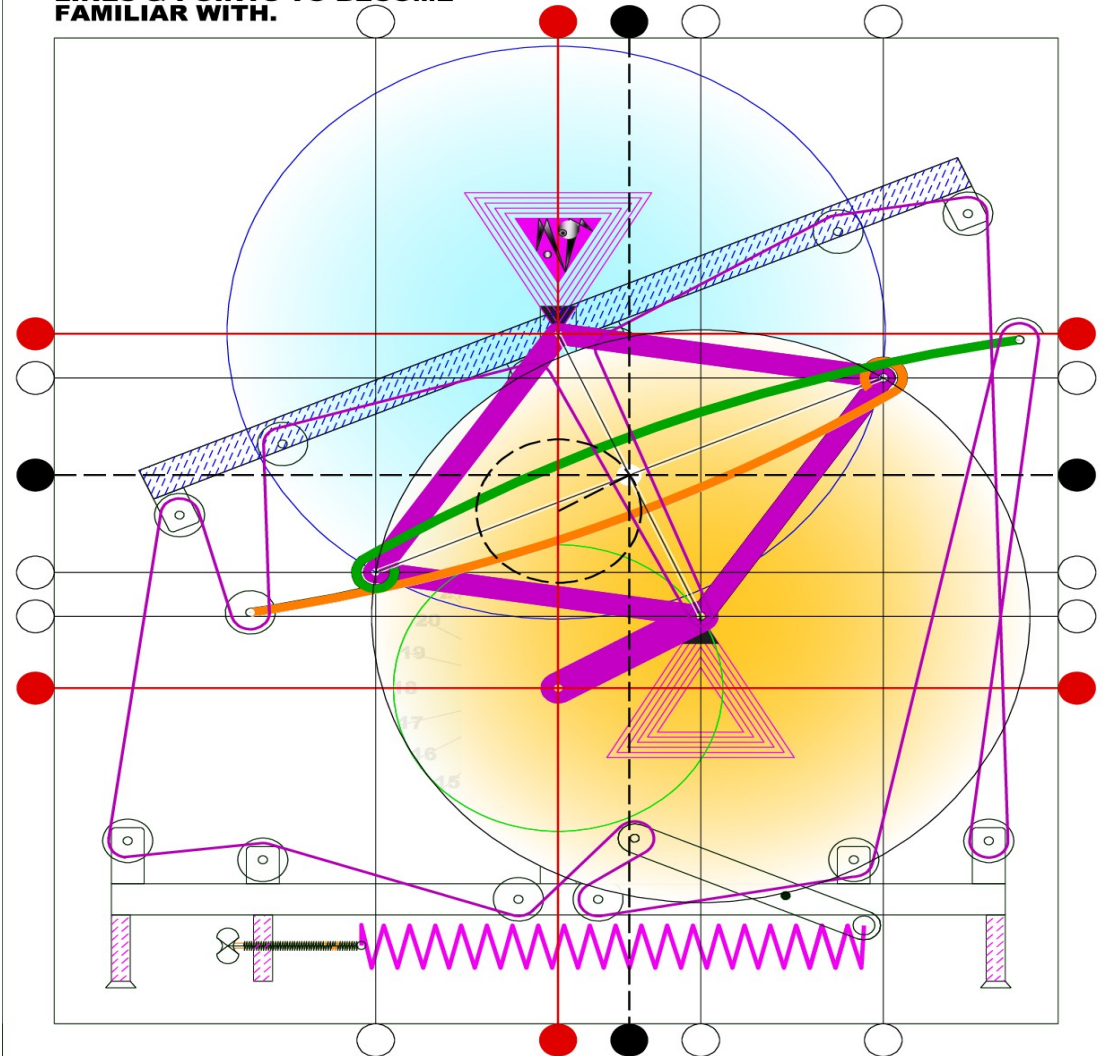
UNLESS
MECHANICAL
BREAKDOWN
OCCUR,
THERE
SHOULD
BE NO
STOPPING
OF ROTATION.

EB IS SHOWN
IN DIAGRAM.
THE EB CAN
OPERATE
VERTICAL
OR
HORIZONTAL
SINCE IT IS
NOT GRAVITY
DEPENDANT.

TO FOLLOW :

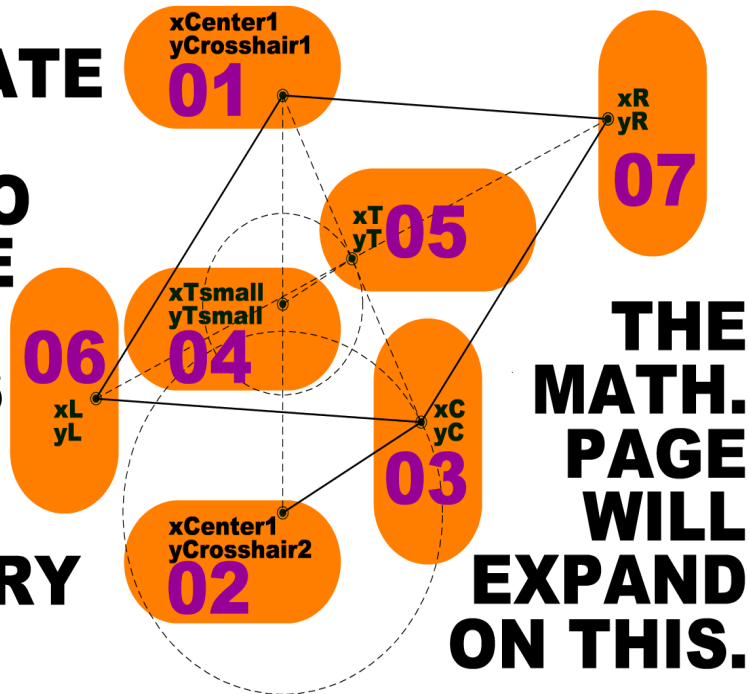
BELOW IS SOME MAJOR GEOMETRY
LINES & POINTS TO BECOME
FAMILIAR WITH.

EB PRACTICAL GENERATOR 01
EB BASIC CALCULATIONS 02
EB PROGRAMMING 03
EB THEORY 04



NOW JUST BEFORE INTRODUCING SOME REAL MATHEMATICAL FORMULA'SLET'S DO A SOFT GEOMETRIC INTRO SUMMATION.
01 & 02 IS CHOSEN CO-ORDINATES.

**CALCULATE
IN 7
STEPS TO
FIND THE
EB
CO-ORDS
AT
MAJOR
GEOMETRY
POINTS**



03, 04, 05, 06 & 07 etc
IS THE CALCULATED
CO-ORDINATES.



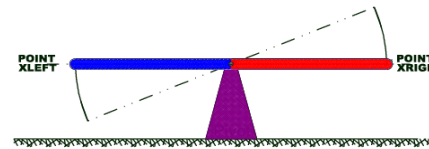
EB_MATH._01

The EB 1/24th TWIN WAVE as a Mechanical Discovery (A SeeSaw Comparison are used as a Simplified Explanation)

Please START here from POINT 01

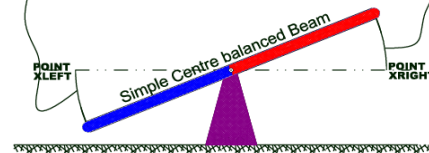
POINT 01 :

DISPLACEMENT on one side is IDENTICAL to the Displacement on the other side even though it take place in a REVERSE MIRROR fashion.



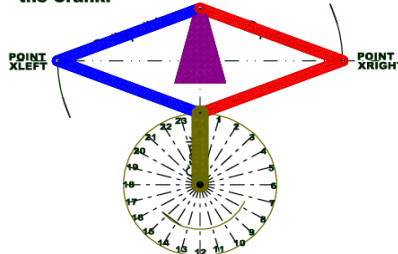
POINT 02 :

So to put it another way :- DISPLACEMENT = distance travelled on one end, is the exact same as on the opposite end of the equal length / balanced centre pivoted seesaw.



POINT 03 :

NOW lets change the simple beam to a Collapsible Square fixed to a Crank (or Circle) and put motion on the whole system through the Crank.



POINT 04 : The 1/24th discovery

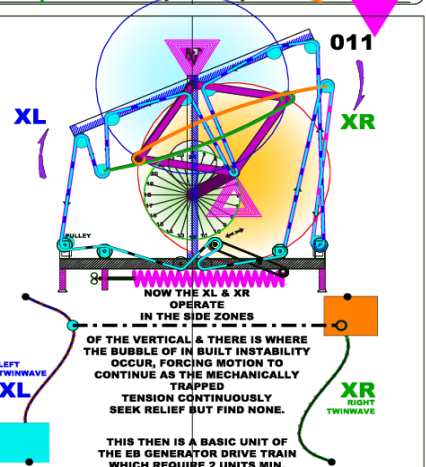
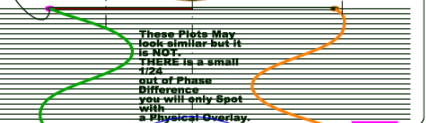
NOW we discover that the DISPLACEMENT on one side (XLEFT as opposed to XRIGHT) is NOT IDENTICAL to the Displacement on the other side while the rest of the Motion also take place in a Obvious REVERSE MIRROR fashion. ONE POINT is 1/24th OUT OF PHASE or BECOME (JUMP actually) 1/24 OUT OF STEP with the opposite POINT.

This Phase difference is easily proven

by plotting XLEFT & XRIGHT as done Below.



Suggestion : USE a CAD Program so as to easily Compare the Red TO Blue Graph. See then the 1/24th Step Difference.



1/24th OUT OF STEP USE:-
HALF OF EB DRIVE TRAIN SHOWN.

EB_MATH._02

0 The EB 1/24th TWIN WAVE Calculations PAGE 01
This is the EB Calculation full page
This will be broken up into Simpler sections
per page that will show how the Calculations add up
to this Full page.

Reposted : 17 FEB 2017

01 BELOW : GEOMETRIC DEDUCTIONS TO GET TO THE FREE POINTS CO-ORDINATES

02 $\pi = 3.1415926535897932$

03 Converting Degrees to Radians
 $\text{angle1} = (\text{AngCR} * (\pi / 180))$ NB : angle1 shown as θ

04 the a lenght of Triangle of Crank Swing
 $a = (\text{CrankR} * \sin(\text{angle1}))$

05 b lenght of Triangle of Crank Swing
 $b1 = (\text{CrankR} * \cos(\text{angle1}))$

06 X coordinate of Crank Swing
 $xC(i) = xCenter1 + a$

07 Y coordinate of Crank Swing
 $yC(i) = yCrosshair2 - b1$

08 Triangle in Square Defined
 $b2 = yCrosshair2 - b1 - yCrosshair1$

09 Side of Top Triangle
 $C2 = \sqrt{a^2 + b2^2}$

10 Half of Side/Top Triangle
 $C3 = C2 / 2$

11 Opposite Angle
 $\text{angle2} = (\text{CrankR} * \sin(\text{angle1})) / C2$

12 Half of Chord
 $W = \sqrt{C2^2 - C3^2}$

13 BELOW : CALCULATING CO-ORDINATES FOR TRACE CIRCLE
 (needed for FREE POINT Calc)

14 x Distance to trace circle centre
 $xTsmall = C3 * \sin(\text{angle2})$

15 y Distance to trace circle centre
 $yTsmall = C3 * \cos(\text{angle2})$

16 x Value for Trace Circle swing
 $xT(i) = xCenter1 + xTsmall$

17 y Value for Trace Circle swing
 $yT(i) = yCrosshair1 + yTsmall$

18 BELOW : CALCULATING CO-ORDINATES FOR SQUARE LEVER : 2 SETS OF X & Y CO-ORDINATES

19 $xL(i) = xT(i) - (W * \cos(\text{angle2}))$

20 $yL(i) = yT(i) + (W * \sin(\text{angle2}))$

21 $xR(i) = xT(i) + (W * \cos(\text{angle2}))$

22 $yR(i) = yT(i) - (W * \sin(\text{angle2}))$

23 NOW TO CALCULATE PENDULUM EXTENSIONS: x & y POINTS
 DEPENDANT ONLY ON VARIABLES w & c3 & angle2.

24 $cbig = \sqrt{Rbig^2 - W^2}$

25 $aBigx = (C3 + cbig) * \sin(\text{angle2})$

26 $bBigy = (C3 + cbig) * \cos(\text{angle2})$

27 $xE(i) = xCenter1 + aBigx$

28 $yE(i) = yCrosshair1 + bBigy$

11 $\psi = R * \sin \theta$

12 INTERMEDIATE PURPOSE : TO GET W
 $W = \sqrt{S^2 - C3^2}$ or $W = \sqrt{S^2 - (\frac{V}{2})^2}$
 $H = 2 \left(\sqrt{S^2 - (\frac{V}{2})^2} \right)$

08 $b2 = \text{Dist 1} - b1$

05 $b1 = R * \cos \theta$

10 $C3 = \frac{V}{2}$

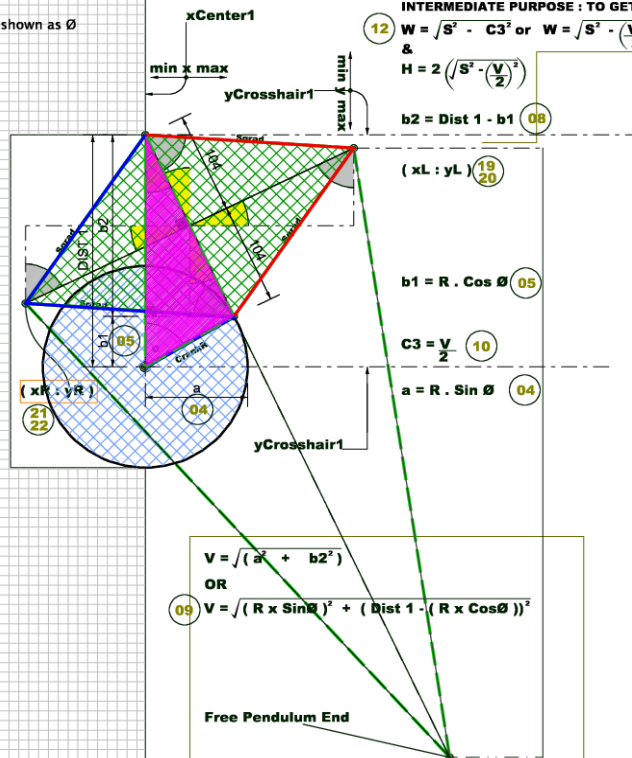
04 $a = R * \sin \theta$

09 $V = \sqrt{(R * \sin \theta)^2 + (\text{Dist 1} - (R * \cos \theta))^2}$

30 ULTIMATE PURPOSE : TO GET Dist 2
 $\text{Dist 2} = (\sqrt{Rbig^2 - W^2}) + C3$

29 ADDITIONAL CALC FOR Dist2 variable
 Dist2 is the Distance between the Pivot to the EB pendulums free point.
 This can be used to determine if any true curve are followed by the free point.
 09 Vertical EB prove that there are no true curve followed. The Determination of Lopsidedness in the Closed Loop Or full Symmetry of this Loop this need to be made.
 This is needed so that if one would use this EB Pendulum as a Mathematical analogy for the 4 acknowledged forces in Nature with the Pendulum Extension Loop as the Representation for Gravity - then in the case of a Lopsidedness in such a Loop - such Lopsidedness would Point to an inherent Directionality inbuilt into Gravity thus giving Rise on the Larger Scale to a Specific Direction of Motion in Space of all Things Gravitationally Linked.

30 $\text{Dist2}(i) = (\sqrt{Rbig^2 - W^2}) + C3$



AND THERE IT IS..STARTING
 THE MUCH-ANTICIPATED.....

EB_MATH._03

0 The EB 1/24th TWIN WAVE as a Mechanical Discovery
This is the EB Calculation 1st breaking page
This will start to show with a Simpler section
that will indicate how the Calculations add up
to the Full page. We start with the
1st step : the ROTATION CALCULATION.

Reposted : 17 FEB 2017

By the way :- this is the co-ordinate system used by the EB Twinwave Twinesine program graphics.

1st STEP

01 BELOW : GEOMETRIC DEDUCTIONS TO GET TO CRANK ROTATION'S xC : yC CO-ORDINATES.

02 $\pi = 3.1415926535897932$

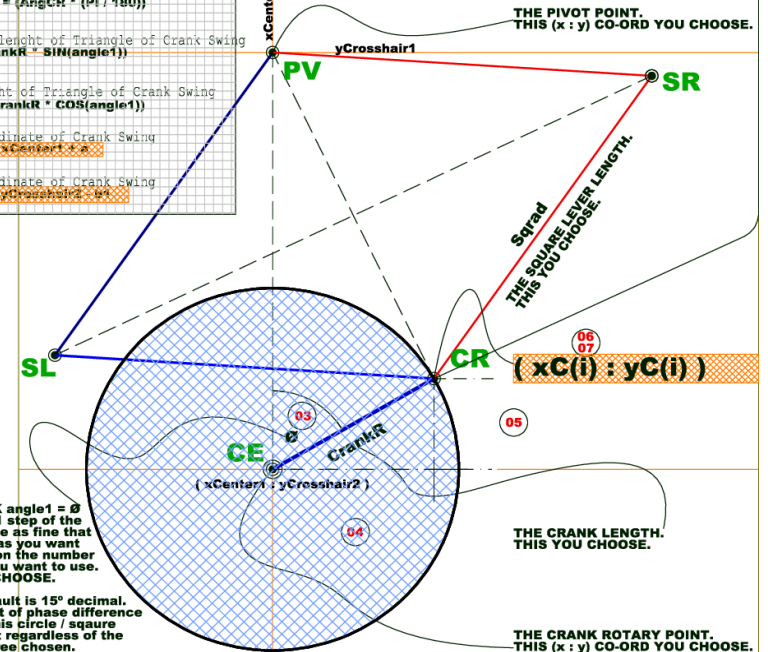
03 Converting Degrees to Radians
 $\text{angle1} = (\text{AngCR} * (\pi / 180))$

04 the a lenght of Triangle of Crank Swing
 $a = (\text{CrankR} * \sin(\text{angle1}))$

05 b lenght of Triangle of Crank Swing
 $b1 = (\text{CrankR} * \cos(\text{angle1}))$

06 X coordinate of Crank Swing
 $xC(i) = xCenter1 + a$

07 Y coordinate of Crank Swing
 $yC(i) = yCrosshair2 - b1$



THE CRANK angle1 = θ
 THIS angle1 step of the crank can be as fine that is as many as you want depending on the number of arrays you want to use. THIS YOU CHOOSE.

The EB default is 15° decimal.
 The 1/24 out of phase difference remain in this circle / square arrangement regardless of the angle1 degree chosen.

THE CRANK ROTARY POINT.
 THIS (x : y) CO-ORD YOU CHOOSE.

This (xC(i) : yC(i)) will fill up it's pre-defined arrays; now after reaching goal:- 06+07 :- on to 2nd step.

- 0 The EB 1/24th TWIN WAVE as a Mechanical Discovery 
 This is the EB Calculation 2nd breakup page.
 The goal here is to determine the dynamically changing angle2 or Ψ , by getting distance C2 or V.

Reposted : 17 FEB 2017

NOW THAT THE SIMPLE ROTATION ARRAY CO-ORDS FOR CRANK POINT CR - ARE FILLED IN GOAL 6 + 7, WE USE THAT ARRAY TO PROGRESS TO GOAL 8.

08 09 (xC(i) : yC(i))
 (xG(1) : yG(1))
 (xG(2) : yG(2))
 (xG(3) : yG(3))
 (xG(4) : yG(4))
 (xG(5) : yG(5))
 (xC(etc) : yC(etc))

b2 = yCrosshair2 - b1 - yCrosshair1

08 Triangle in Square Defined

09 C2= SQR(a ^ 2 + b2 ^ 2)

09 Side of Top Triangle

C3 = C2 / 2

10 Half of Side/Top Triangle [FOR LATER USE]

11 Opposite Angle
 $\text{angle2} = (\text{CrankR} * \sin(\text{angle1})) / C2$
 or




SEE ALSO TRIG. 101 BELOW :

THE VARIETY OF FORMULAS GIVEN ABOVE FOR THE SAME GOAL WILL HELP TO TRAIN THE MIND TO REGONIZE THAT YOU CAN USE WHATEVER DESIGNATION FOR VARIABLES THAT SUITS YOU IN YOUR OWN PROGRAMMING. ONLY THE FORMULA PATTERN NEEDS TO BE UNDERSTOOD.

THE CRANK angle1 = θ
 $a = (\text{CrankR} * \sin(\text{angle1}))$

TRIGONOMETRY 101 : PLEASE SEE <http://mathworld.wolfram.com/Trigonometry.html>

TRIGONOMETRY 101 : PLEASE SEE <https://www.mathsisfun.com/geometry/radians.html>
 GOAL 11: EMPLOY A STANDARD TRIGONOMETRY FORMULA FOR DETERMINING AN UNKNOWN ANGLE IN A 90 ° TRIANGLE WHERE 2 SIDES ARE KNOWN:- LEAVING US WITH THE GOAL OF FINDING 1 ANGLE = ψ .

$\sin \psi = \frac{O}{h}$  $\cos \psi = \frac{a}{h}$  $\tan \psi = \frac{O}{a}$ 

This angle2 or Ψ , will fill up it's pre-defined arrays;- now after reaching goal:- 11 ;- on to 3rd step.

REMINDER : THE SPECIALL CASE OF A TRIANGLE WITH A 90° DEGREE CORNER LEND US TO MAKE USE OF THE SIMPLE RATIO (SIMPLE DIVISION) OF THE SIDE LENGHTS OF THIS TRIANGLE TO MAP TO A LIST OF DEGREES CORRESPONDING TO THE CLASICAL 360 DEGREES SYSTEM OR AS IN THIS CASE OF USE, - THE RADIAN SYSTEM.

(THE LIST IS BUILD IN AS A MATHEMATICAL FUNCTION IN YOUR COMPUTER / CALCULATOR)

SEE ALSO TRIG. 101 BELOW :

2nd STEP

$\psi = R * \sin \theta = \text{angle2}$

V

10

11

08

09

10

17

radius = r and length = r

1 Radian = is about 57.2958 decimal degrees

radius = r

length

radius = r

length

radius = r

length

radius = r


length

radius = r

length

radius = r

length

- 0 The EB 1/24th TWIN WAVE as a Mechanical Discovery 
 This is the EB Calculation 3rd breakup page.
 The goal here is to determine the dynamically changing distance W;- to get to calculate TC point's xT:yT Co-ords.

Reposted : 17 FEB 2017

12 $W = \text{SQR}(\text{Sqr}a^2) - (C3^2)$
 Half of Chord

13 BELOW : CALCULATING CO-ORDINATES
 (FOR THE TRACE POINT Calc)

xTsmall = C3 * SIN(angle2)

14 x Distance to trace circle centre

yTsmall = C3 * COS(angle2)

15 y Distance to trace circle centre

16 x Value for Trace Circle swing

17 y Value for Trace Circle swing

REVEALING THE TRACE CIRCLE

The W- Distance will be used with the xT(i) & yT(i) to get to SL & SR co-ordinate values.

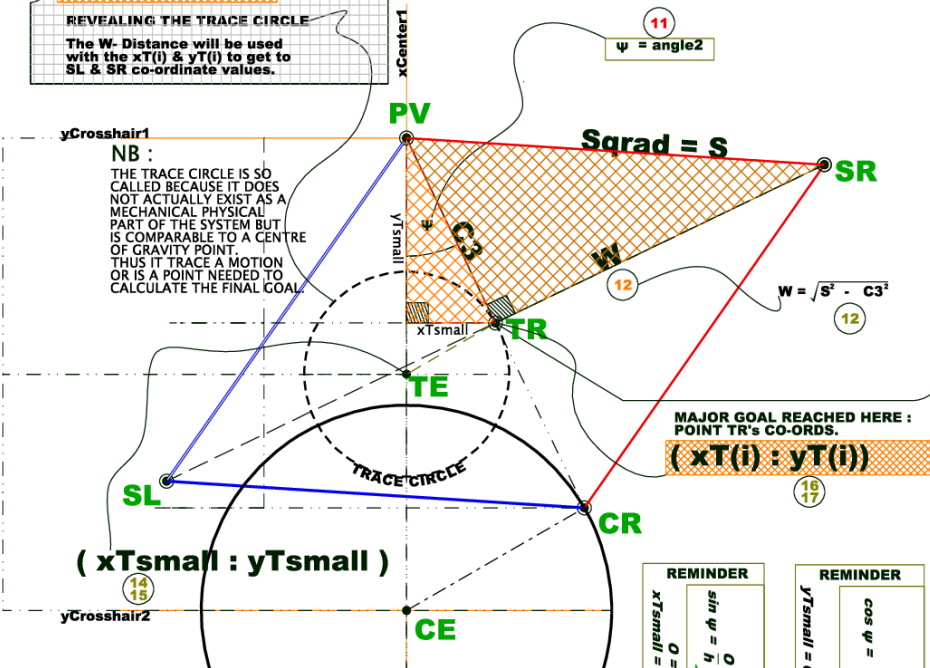
REMINDER : THE TRACE CIRCLE PLACEMENT IN MOST CASES IS JUST 1/2 OF THE PIVOT TO CRANK CENTRE DISTANCE.

$a^2 = b^2 + c^2$

REMINDER : Note the Heavy use of the Pythagoras formula throughout the EB Twinwave Twinsine Program.

$a = \sqrt{b^2 + c^2}$

3rd STEP



NB : TO CALCULATE THE FINAL GOAL.
 FROM THE TR = TRACE CENTRE, CENTRE POINT CO-ORDS, THE SL & SR - POINT CO-ORDS CAN BE CALCULATED - THAT BEING THE ULTIMATE GOAL.

THAT SL & SR - POINTS CAN THEN BE USED TO PLOT THE TWINWAVE GRAPHS, THAT WILL REVEAL THE 1/24th OUT OF PHASE / OUT OF STEP MECHANICAL HARDWIRED QUALITY OF THIS PHYSICAL SYSTEM.

REMINDER
 $\sin \psi = \frac{O}{h}$
 $O = h * \sin \psi$
 $xTsmall = C3 * \sin(\text{angle2})$

REMINDER
 $\cos \psi = \frac{a}{h}$
 $a = h * \cos \psi$
 $yTsmall = C3 * \cos(\text{angle2})$

The xT(i) & yT(i) will fill up it's pre-defined arrays;- now after reaching goal:- 16+17 ;- on to 4th step.

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The EB 1/24th TWINWAVE as a Mechanical Discovery

This is the EB Calculation 6th breakup page.

The goal here is to show the extension point of the EB pendulum calculation.

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23 NOW TO CALCULATE PENDULUM EXTENSIONS: x & y POINTS
DEPENDANT ONLY ON VARIABLES w & c3 & angle2.

24 $cbig = \text{SQRT}(Rbig^2 - W^2)$

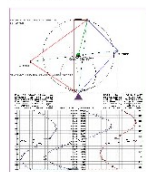
25 $aBigx = (C3 + cbig) * \sin(\text{angle2})$

26 $bBigy = (C3 + cbig) * \cos(\text{angle2})$

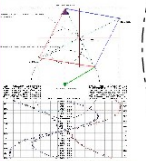
27 $yE(i) = yCrosshair1 + bBigy$

28 $xE(i) = xCenter1 + aBigx$

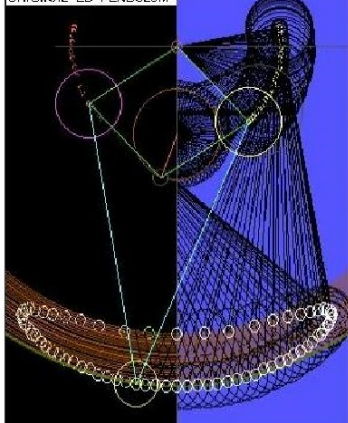
eb"elastic"
example



eb"realistic"
example



ORIGINAL "EB" PENDULUM



ULTIMATE PURPOSE: TO GET Dist 2
 $\text{Dist 2} = (\sqrt{Rbig^2 - W^2}) + C3$

(xE(i) : yE(i))

Now that we got the xE + yE Co-ords we go to a Summation page on as a 7th step. That will conclude the EB formula.

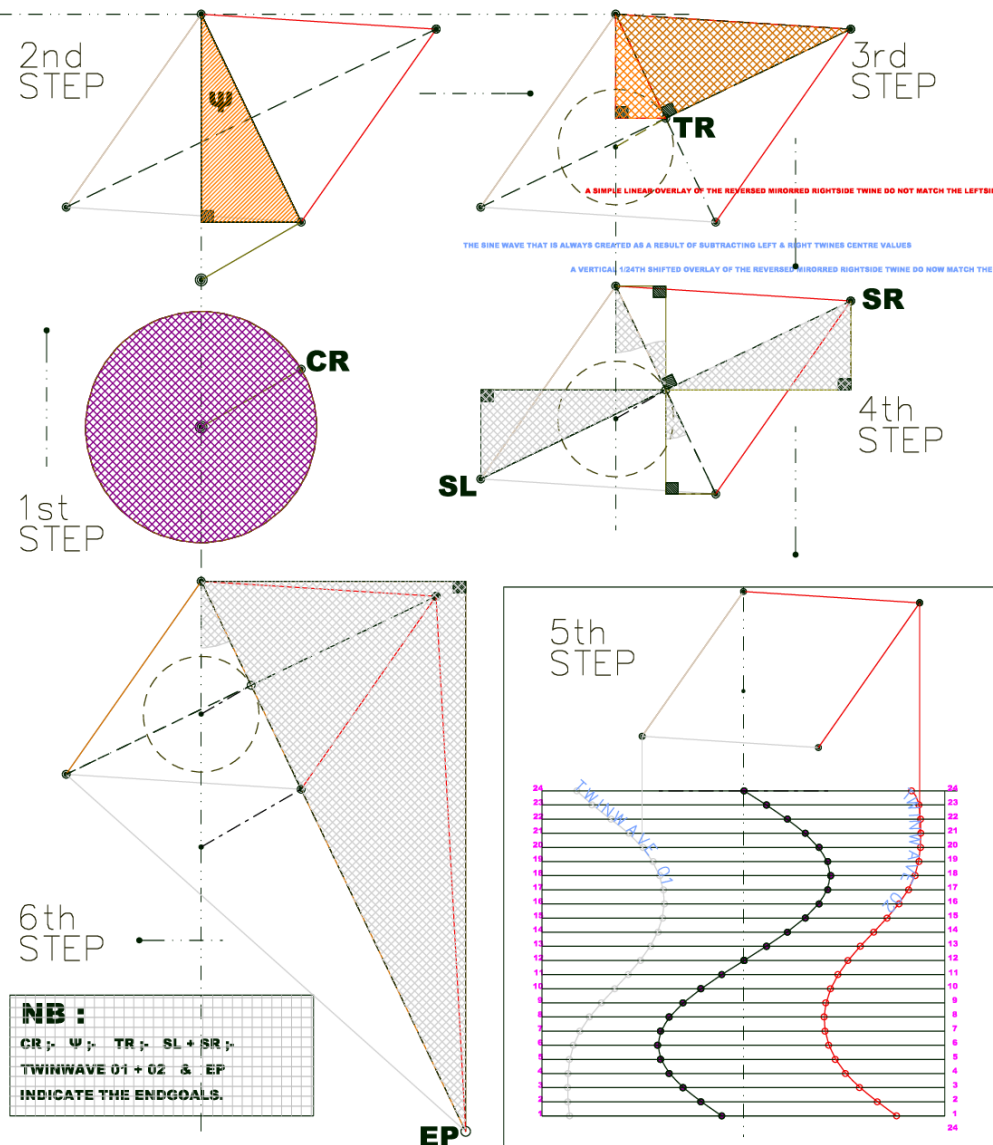
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The EB 1/24th TWINWAVE as a Mechanical Discovery

This is the EB Calculation 7th breakup page.

The goal here is to show the EB formula's in a single glance as a series of Pythagoras + Trigonometry application to Triangles.

Reposted : 17 FEB 2017



Now that we got the xE + yE Co-ords we go to a Summation page as a 7th step. This then conclude the EB formula.

And here at last, the Maths turn
NASTY.....very much so indeed.

But calm down... you only need to
really deal with the below setup in
your calculations / programming of the eb, as will be
needed for effective use.

Pg018

